

Automatic Control

If you have a smart project, you can say "I'm an engineer"

Lecture 2

Staff boarder

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Automatic Control

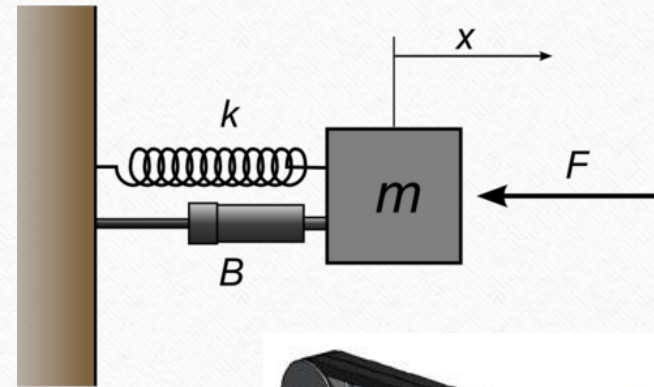
MPE 424

- **Lecture aims:**
 - Understand the mathematical modeling of mechanical systems and hydraulic systems.

Basic Types of Mechanical Systems

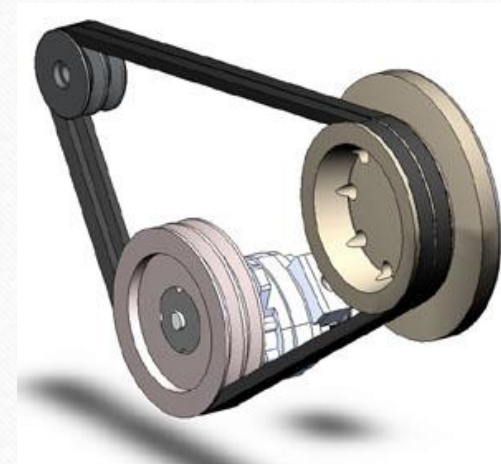
- Translational

- Linear Motion



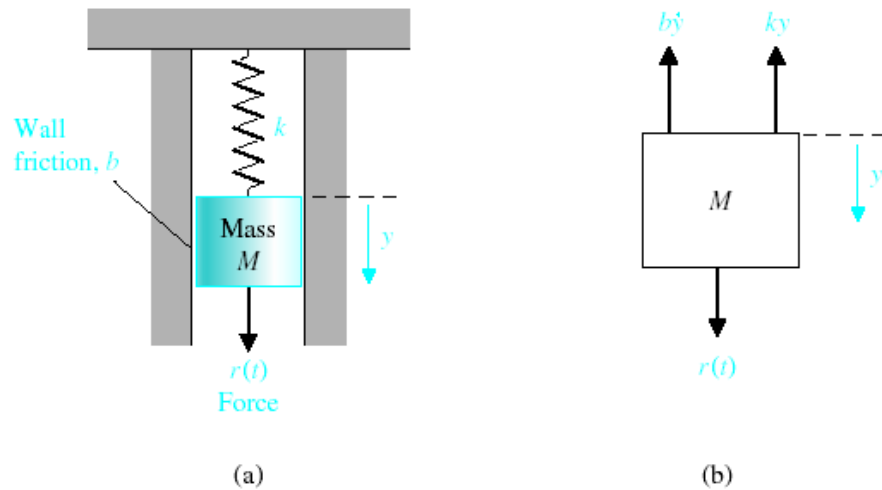
- Rotational

- Rotational Motion



Modeling of Mechanical System

- **Spring - Mass - Damper**

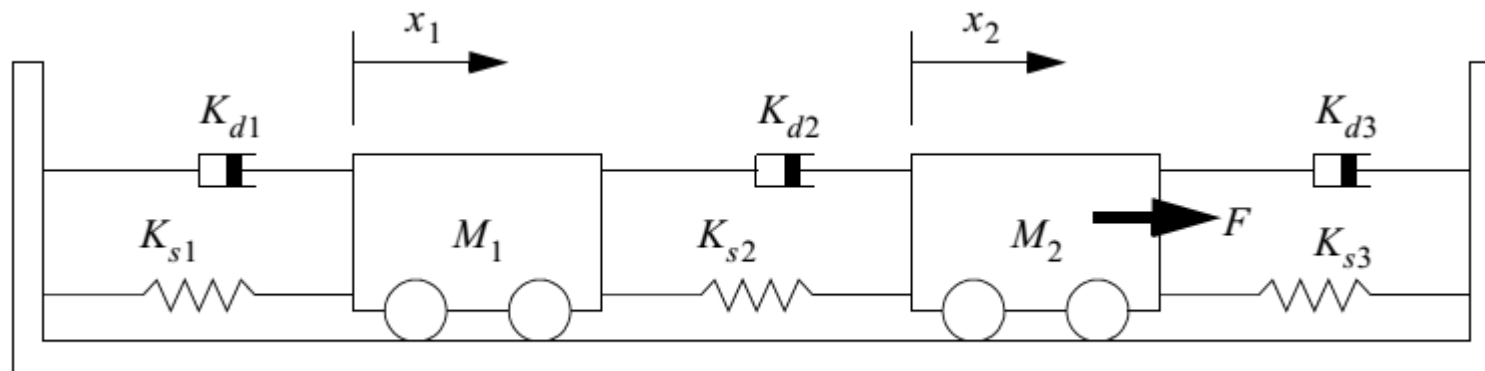


$$M \cdot \frac{d^2}{dt^2} y(t) + b \cdot \frac{d}{dt} y(t) + k \cdot y(t) = r(t)$$

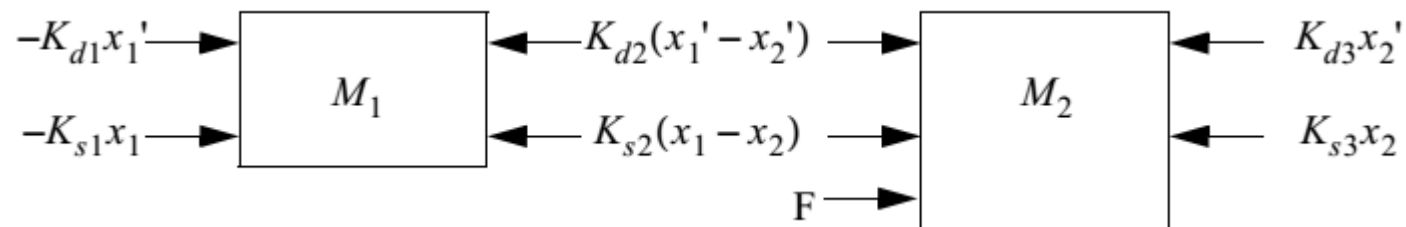
(a) Spring-mass-damper system.
(b) Free-body diagram.

Modeling of Mechanical system

Mathematical Models for the Schematic



- Free Body Diagram FBD

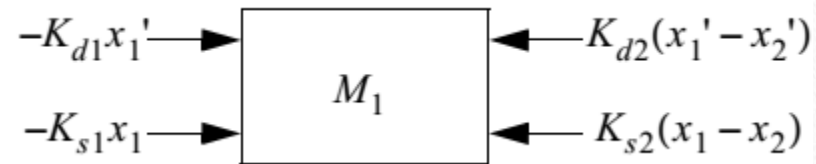


Modeling of Mechanical system

Write equation of motion: Two degree of freedom

Assume $X_1 > X_2$ positive direction of motion \rightarrow

- For mass(1)



$$\sum F = -K_{d1}x_1' - K_{s1}x_1 - K_{d2}(x_1' - x_2') - K_{s2}(x_1 - x_2) = M_1x_1''$$

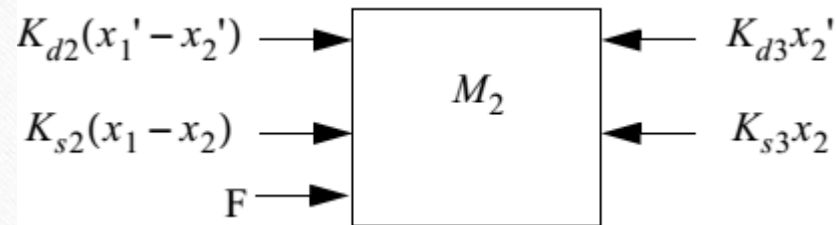
$$x_1''(M_1) + x_1'(K_{d1} + K_{d2}) + x_1(K_{s1} + K_{s2}) + x_2'(-K_{d2}) + x_2(-K_{s2}) = 0$$

Modeling of Mechanical system

Write equation of motion: Two degree of freedom

Assume $x_1 > x_2$ positive direction of motion \rightarrow

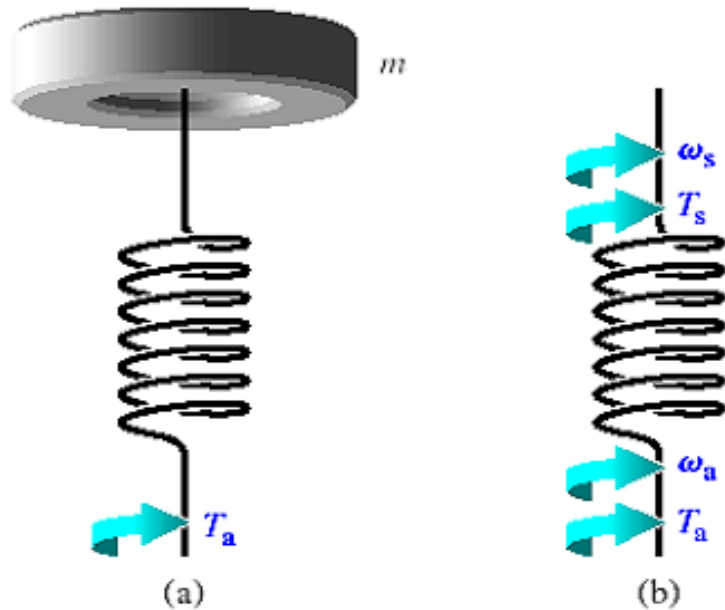
- For mass(2)



$$\sum F = K_{d2}(x_1' - x_2') + K_{s2}(x_1 - x_2) + F - K_{d3}x_2' - K_{s3}x_2 = M_2x_2''$$

$$x_2''(M_2) + x_2'(K_{d2} + K_{d3}) + x_2(K_{s2} + K_{s3}) + x_1'(-K_{d2}) + x_1(-K_{s2}) = F$$

Modeling of Mechanical System



(a) Torsional spring-mass system.

(b) Spring element.

$$T_a(t) - T_s(t) = 0$$

$$T_a(t) = T_s(t)$$

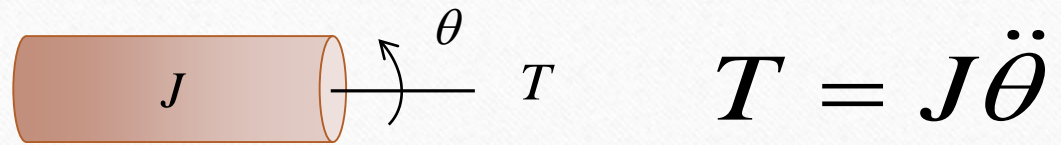
$$\omega(t) = \omega_s(t) - \omega_a(t)$$

$T_a(t)$ = through - variable

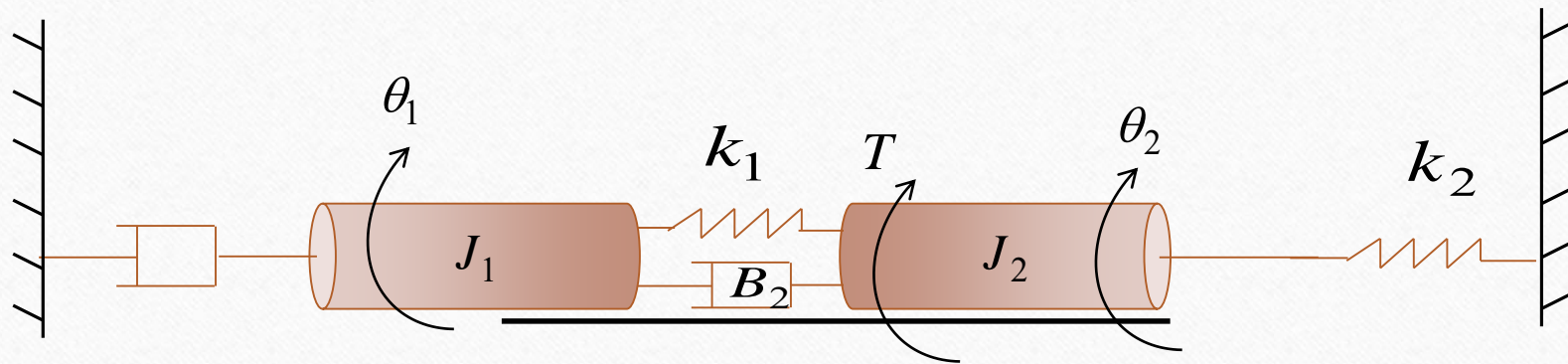
angular rate difference = across-variable

Example

Moment of Inertia



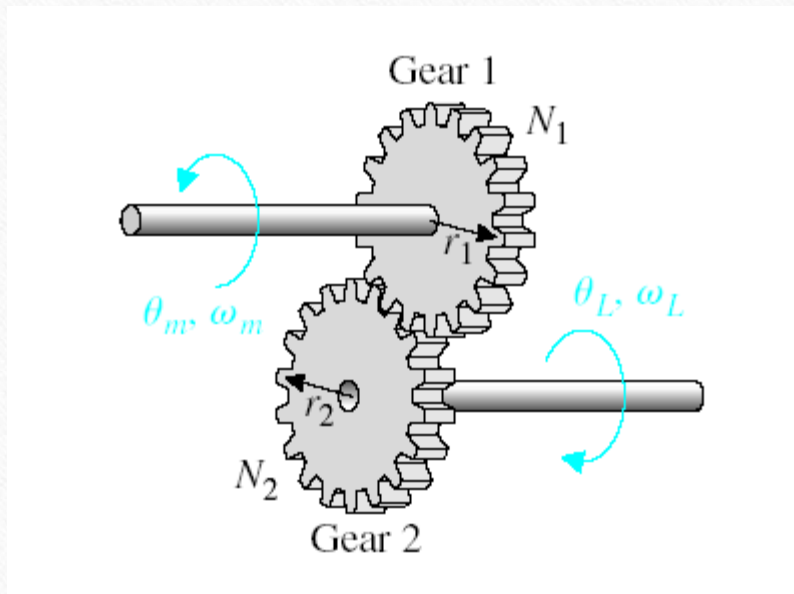
$$T = J\ddot{\theta}$$



Mechanical Building Blocks

Building Block	Equation
	Translational
Spring	$F = kx$
Damper	$F = c \, dx/dt$
Mass	$F = m \, d^2x/dt^2$
	Rotational
Spring	$T = k\theta$
Damper	$T = c \, d\theta/dt$
Moment of inertia	$T = J \, d^2\theta/dt^2$

The Transfer Function of Linear Systems



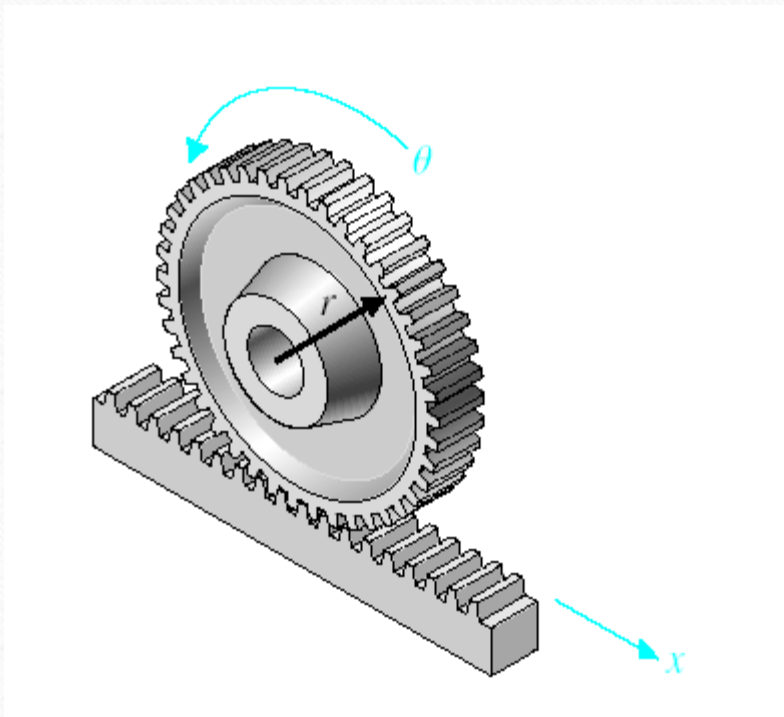
$$\text{Gear Ratio} = n = N_1/N_2$$

$$N_2 \cdot \theta_L = N_1 \cdot \theta_m$$

$$\theta_L = n \cdot \theta_m$$

$$\omega_L = n \cdot \omega_m$$

The Transfer Function of Linear Systems



$$x = r \cdot \theta$$

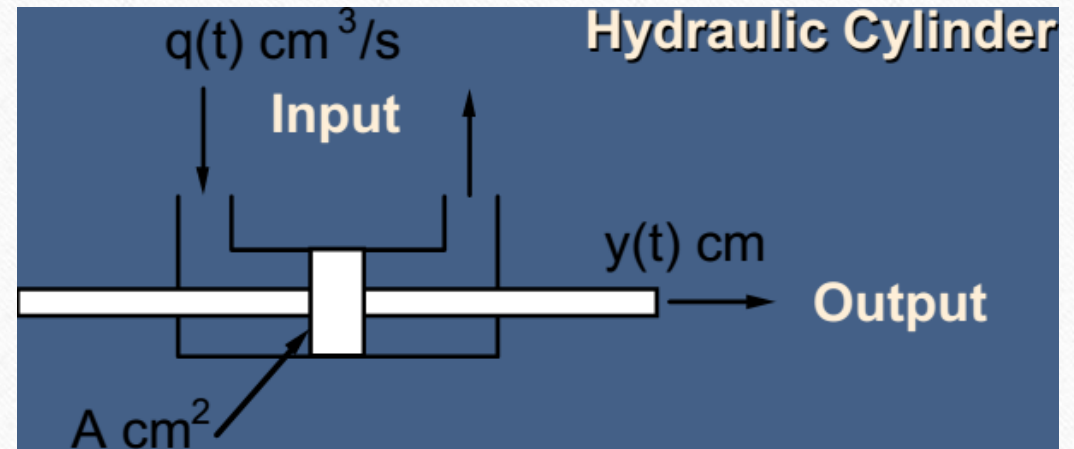
converts radial motion to linear motic

Modeling of Hydraulic System

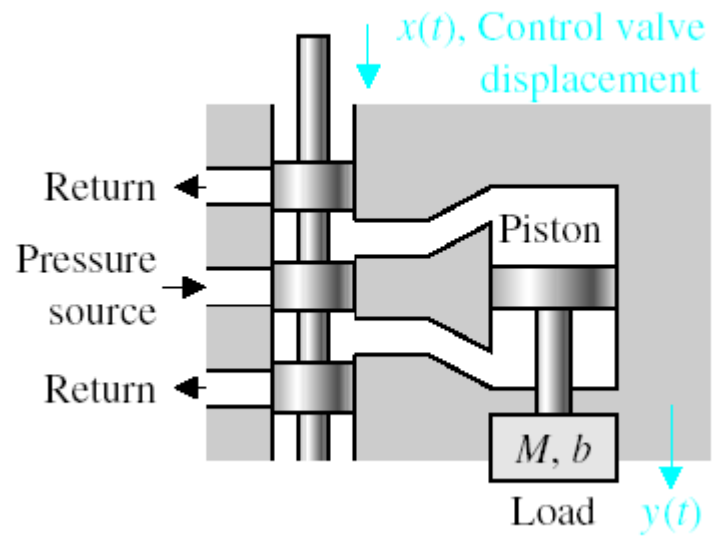
Continuity equation

$$A \frac{dy(t)}{dt} = q(t)$$

$$\frac{dy(t)}{dt} = Kq(t)$$



The Transfer Function of Linear Systems



$$\frac{Y(s)}{X(s)} = \frac{K}{s(Ms + B)}$$

$$K = \frac{A \cdot k_x}{k_p} \quad B = \left(b + \frac{A^2}{k_p} \right)$$

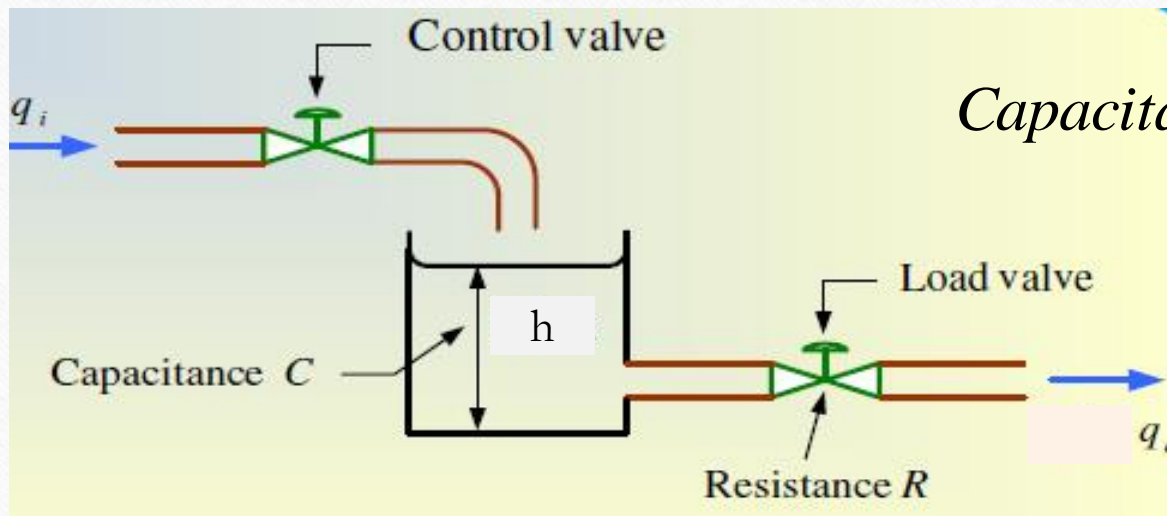
$$k_x = \frac{d}{dx} g \quad k_p = \frac{d}{dP} g$$

$$g = g(x, P) = \text{flow}$$

$$A = \text{area of piston}$$

Modeling of Hydraulic System

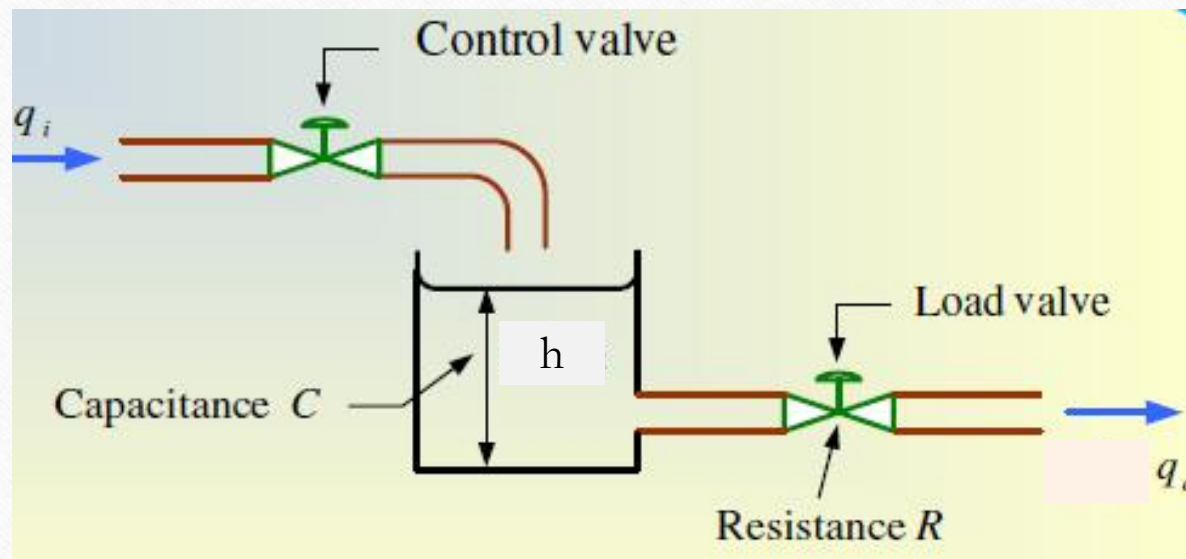
- The capacitance of a tank is defined to be the change in quantity of stored liquid necessary to cause a unity change in the height.



$$\text{Capacitance} = \frac{\text{change in liquid stored}}{\text{change in height}} = \frac{m^3}{m} \text{ or } m^2$$

- Capacitance (C) is cross sectional area (A) of the tank.

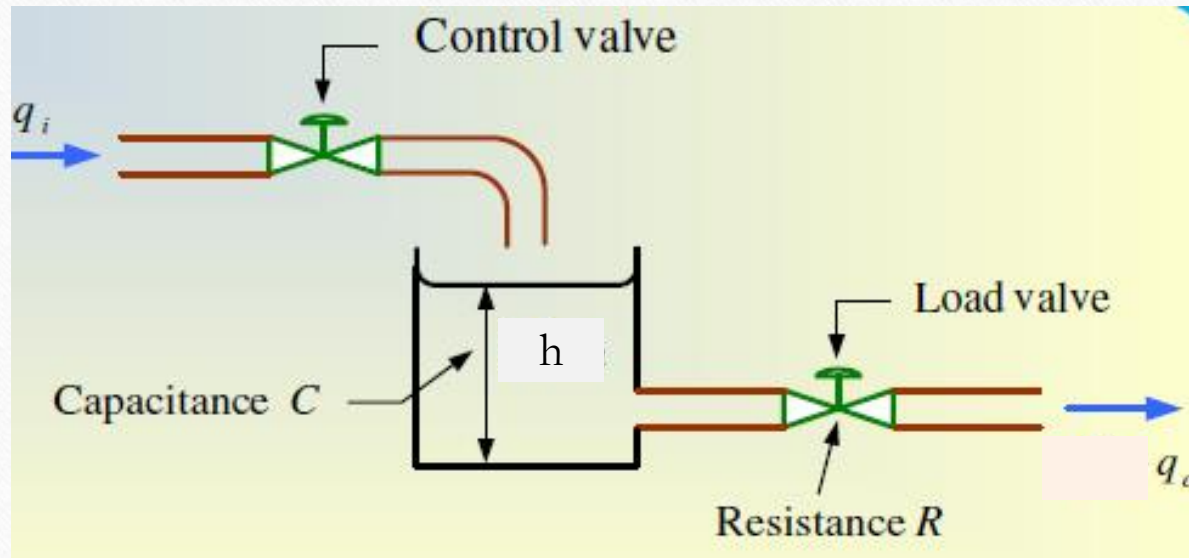
Capacitance of Liquid-Level Systems



Rate of change of fluid volume in the tank = flow in – flow out

$$\frac{dV}{dt} = q_i - q_o \quad \frac{d(A \times h)}{dt} = q_i - q_o$$

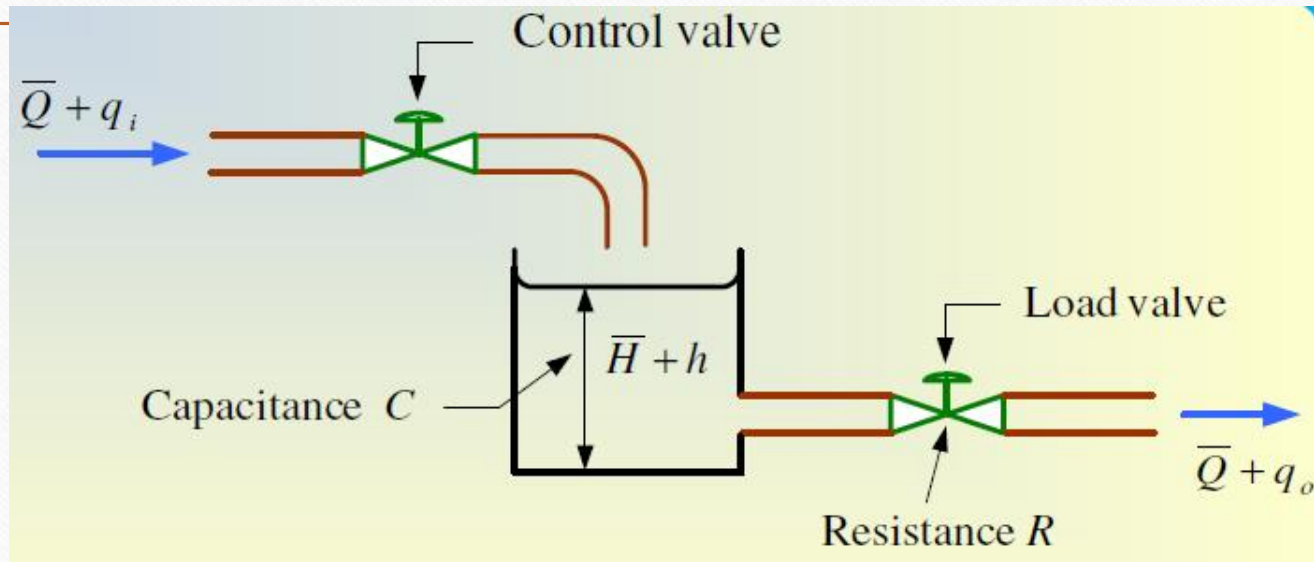
Capacitance of Liquid-Level Systems



$$A \frac{dh}{dt} = q_i - q_o$$

$$C \frac{dh}{dt} = q_i - q_o$$

Modelling Example



\bar{H} = steady-state head (before any change has occurred), m.

h = small deviation of head from its steady-state value, m.

\bar{Q} = steady-state flow rate (before any change has occurred), m^3/s .

q_i = small deviation of inflow rate from its steady-state value, m^3/s .

q_o = small deviation of outflow rate from its steady-state value, m^3/s .

Modelling Example

- The rate of change in liquid stored in the tank is equal to the flow in minus flow out.

$$C \frac{dh}{dt} = q_i - q_o \longrightarrow (1)$$

- The resistance R may be written as

$$R = \frac{dH}{dQ} = \frac{h}{q_o} \longrightarrow (2)$$

- Rearranging equation (2)

$$q_o = \frac{h}{R} \longrightarrow (3)$$

Modelling Example

$$C \frac{dh}{dt} = q_i - q_o \longrightarrow (1) \quad q_o = \frac{h}{R} \longrightarrow (4)$$

- Substitute q_o in equation (3)

$$C \frac{dh}{dt} = q_i - \frac{h}{R}$$

- After simplifying above equation

$$RC \frac{dh}{dt} + h = Rq_i$$

- Taking Laplace transform considering initial conditions to zero $RCsH(s) + H(s) = RQ_i(s)$

Modelling Example

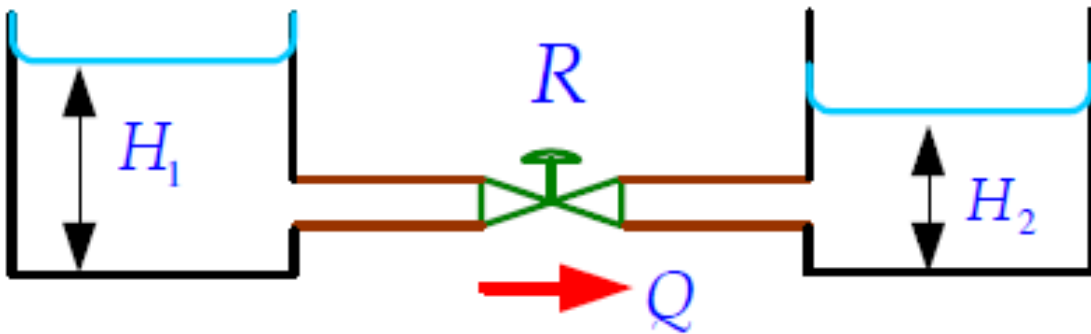
$$RCsH(s) + H(s) = RQ_i(s)$$

- The transfer function can be obtained as

$$\frac{H(s)}{Q_i(s)} = \frac{R}{(RCs + 1)}$$

Modeling of Hydraulic System

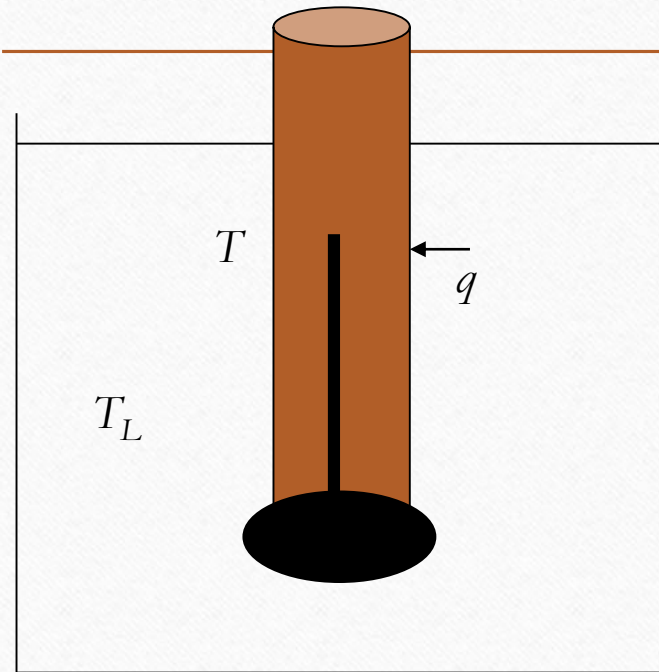
- The resistance for liquid flow in such a pipe is defined as the change in the level difference necessary to cause a unit change inflow rate.



$$\text{Resistance} = \frac{\text{change in level difference}}{\text{change in flow rate}} = \frac{m}{m^3 / s}$$

$$R = \frac{\Delta(H_1 - H_2)}{\Delta Q} = \frac{m}{m^3 / s}$$

Modeling of Thermal System



q is the net rate of heat flow

C is the capacitance

R is the thermal resistance

$$q = \frac{T_L - T}{R}$$

$$q_1 - q_2 = C \frac{dT}{dt}; q = C \frac{dT}{dt}$$

$$C \frac{dT}{dt} = \frac{T_L - T}{R}$$

$$RC \frac{dT}{dt} + T = T_L$$



Thank you

Model Examples

- Car parking



Model Examples

- HVAC system

